



AKADEMIA EDUKACYJNO - INŻYNIERSKA

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MATURA STUDIA PRAKTYKA STAŻ PRACA BIZNES PRZEMYSŁ STEAM

KOMPLEKSOWE WSPARCIE EDUKACYJNE ON – LINE NA KAŻDYM ETAPIE KSZTAŁCENIA INŻYNIERSKIEGO

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MECHANIKA II

"DYNAMIKA PUNKTU MATERIALNEGO – KULKA W RURCE"

PROJEKT

Oblicz:

- prędkość liniową
- nacisk
- ugięcie sprężyny

w punktach charakterystycznych toru ruchu poruszającej się kulki.

DANE:

$$m, V_A, \tau, R, \mu, \beta, h_0, c = OK$$

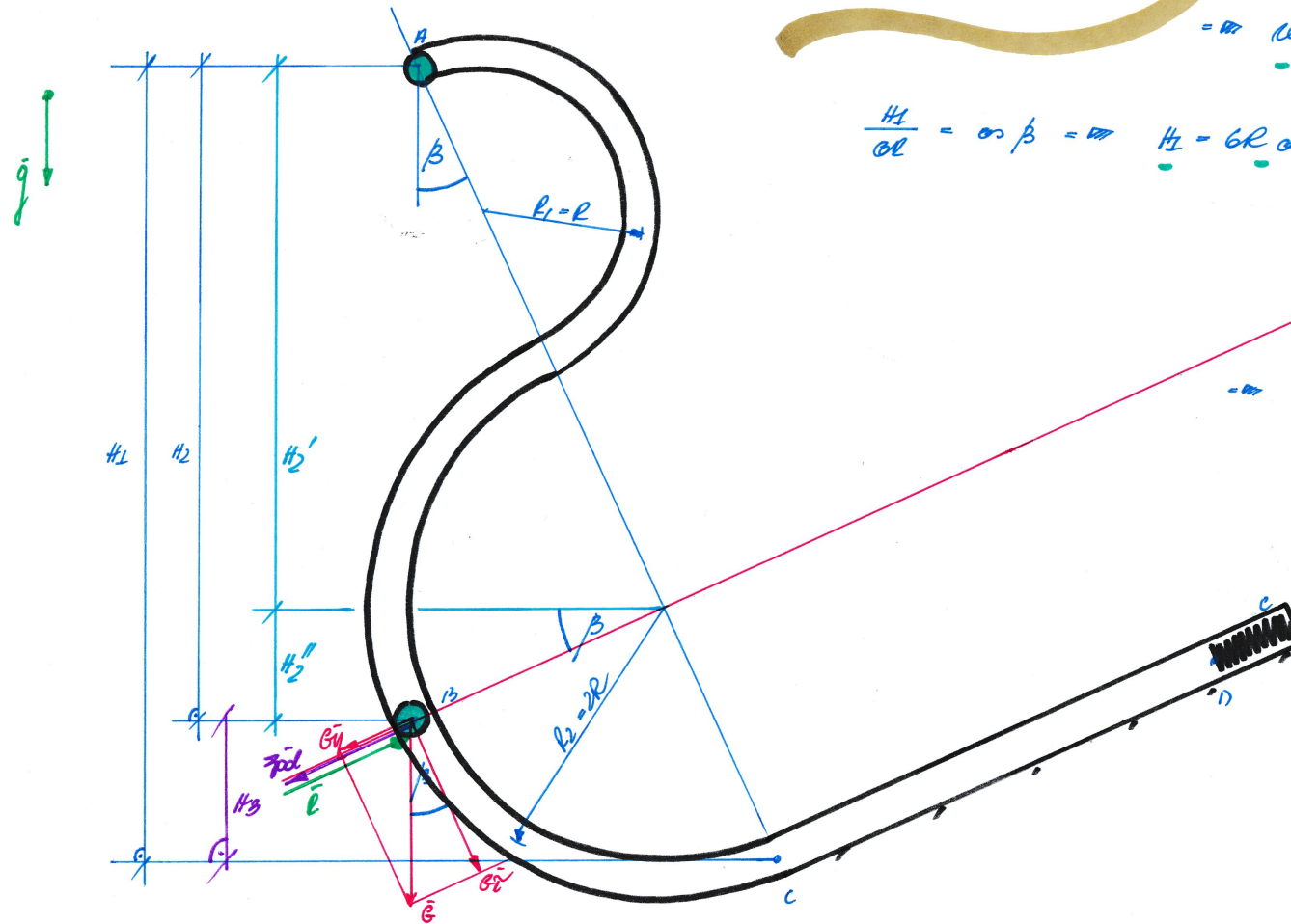
SZUKAM:

$$V_B, V_C, V_D = ?$$

$$N_C = ?$$

$$h = ?$$

21) $\rho: m = 0.5 \text{ kg}$ $\eta = 0.1$
 $v_A = 0.8 \text{ m/s}$ $\beta = 30^\circ$
 $\gamma = 0.1 \text{ s}$ $k_0 = 0$
 $R = 0.2 \text{ m}$ $c = 1000 \text{ N/m}$



$v_A, v_B, v_C = ?$

$N_C = ?$

$k = ?$

$\Delta E_K = \Delta E_P$

$\frac{1}{2} m v_C^2 - \frac{1}{2} m v_A^2 = m g H_2 \cdot 1.2$

$\Rightarrow v_C = \sqrt{v_A^2 + 2g H_2}$

$\Rightarrow v_C = \sqrt{v_A^2 + 2g R \cos \beta}$

$\frac{H_2}{R} = \cos \beta \Rightarrow H_2 = 6R \cos \beta$

$\Rightarrow v_C = \sqrt{(0.8)^2 + 2 \cdot 9.81 \cdot 0.2 \cdot \cos 30^\circ} = 4.68 \text{ m/s}$

115)

$\Delta E_L = \Delta E_P$

$$\frac{1}{2} \cancel{u_0^2} - \frac{1}{2} \cancel{u_0^2} = \cancel{u_0^2} H_2 \quad | \cdot 2$$

$$\underline{u_0} = \sqrt{\underline{u_0^2} + 2gH_2}$$

$H_2 = H_2' + H_2''$

$$\frac{H_2'}{4R} = \sin \beta \quad \Rightarrow \quad \underline{H_2'} = 4R \sin \beta$$

$$\frac{H_2''}{2L} = \sin \beta \quad \Rightarrow \quad \underline{H_2''} = 2L \sin \beta$$

$$\Rightarrow \underline{u_0} = \sqrt{u_0^2 + 4gR(\sin \beta + 2\sin \beta)}$$

$$\begin{aligned} \Rightarrow \underline{u_0} &= \sqrt{(0.8)^2 + 4 \cdot 9.81 \cdot 0.2 (\sin 30^\circ + \sin 30^\circ)} = \\ &= \underline{4.26} \text{ m/s} \end{aligned}$$

116)

$\Delta E_L = -\Delta E_P$

$$\frac{1}{2} \cancel{u_0^2} - \frac{1}{2} \cancel{u_0^2} = -\cancel{u_0^2} H_3 \quad | \cdot 2$$

$$\underline{u_0} = \sqrt{\underline{u_0^2} - 2gH_3}$$

$H_3 = H_1 - H_2$

$$\begin{aligned} H_3 &= 6R \sin \beta - (2L \sin \beta + 4R \sin \beta) = \\ &= 6R \sin \beta - 2L \sin \beta - 4R \sin \beta = \\ &= -2L \sin \beta + 2R \sin \beta = 2(-L \sin \beta + R \sin \beta) \end{aligned}$$

$$\Rightarrow \underline{u_0} = \sqrt{u_0^2 - 4gR(-\sin \beta + \sin \beta)}$$

$$\Rightarrow \underline{u_0} = \sqrt{(4.58)^2 - 4 \cdot 9.81 \cdot 0.2 (-\sin 30^\circ + \sin 30^\circ)} = \underline{4.25} \text{ m/s}$$

$$v_0 = \frac{m v_c - (Gx + T) \tilde{v}}{m}$$

$$\frac{Gx}{G} = m \sin \beta \Rightarrow Gx = mg \sin \beta$$

$$\frac{Gy}{G} = \cos \beta \Rightarrow Gy = mg \cos \beta$$

$$T = \frac{T_1}{T_N} \Rightarrow T = \mu \cdot R = \mu Gy = \mu mg \cos \beta$$

$$\Rightarrow v_0 = \frac{m v_c - \mu (m \sin \beta + \mu \cos \beta) g \tilde{v}}{m}$$

$$v_0 = v_c - (\sin \beta + \mu \cos \beta) g \tilde{v}$$

$$\begin{aligned} v_0 &= 4,68 - (\sin 30^\circ + 0,1 \cos 30^\circ) \cdot 9,81 \cdot 0,1 = \\ &= 4,68 - 0,58 = 4,00 \text{ m/s} \end{aligned}$$

$$\frac{1}{2} T \ddot{x} = m \ddot{x} = -Gx - T$$

$$m \ddot{x} = -mg \sin \beta - \mu mg \cos \beta \quad | : m$$

$$\ddot{x}(t) = -g(\sin \beta + \mu \cos \beta)$$

$$\begin{aligned} \text{I) } \ddot{x}(t) &= A \\ \text{II) } \dot{x}(t) &= At + C_1 \\ \text{III) } x(t) &= \frac{1}{2} At^2 + C_1 t + C_2 \end{aligned}$$

$$\text{I) } \dot{x}(t=0) = v_c$$

$$\text{II) } x(t=0) = x_c = 0$$

$$\text{I) } v_c = A(0) + C_1$$

$$\text{II) } 0 = \frac{1}{2} A(0)^2 + C_1(0) + C_2$$

$$\text{I) } C_1 = v_c$$

$$\text{II) } C_2 = 0$$

$$\text{I) } \ddot{x}(t) = A$$

$$\text{II) } \dot{x}(t) = At + v_c$$

$$\text{III) } x(t) = \frac{1}{2} At^2 + v_c t$$

$\tilde{v}$ = TIME of HAND RELEASE

$$\begin{aligned} \text{I) } \dot{x}(t=\tilde{v}) &= v_0 \\ \text{II) } x(t=\tilde{v}) &= 100 \end{aligned}$$

$$\begin{aligned} \text{I) } v_0 &= A \tilde{v} + v_c \\ \text{II) } 100 &= \frac{1}{2} A \tilde{v}^2 + v_c \tilde{v} \quad (2?) \end{aligned}$$

e)

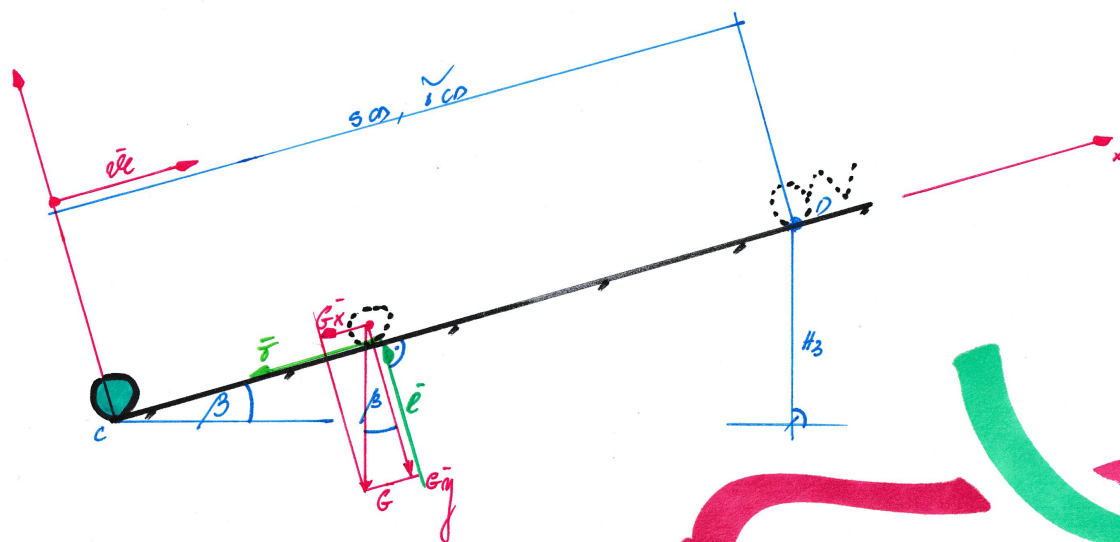
$$\frac{1}{2} \vec{F}_{iq} = m \vec{a}_{iq} = -\vec{G}_x - \vec{T}_{od} + \vec{L}$$

$$0 = -G_y - T_{od} + L$$

$$L = G_y + T_{od}$$

$$\begin{aligned} L &= m g \sin \beta + \frac{m v_{\beta}^2}{l_2} = \\ &= m \left(g \sin \beta + \frac{v_{\beta}^2 + 4 g R (\sin \beta + l \cos \beta)}{2 R} \right) \end{aligned}$$

as)



$$\Delta E_K = -\Delta E_P$$

$$\frac{1}{2} m v_{\beta}^2 - \frac{1}{2} m v_{\alpha}^2 = -m g h_3 \quad | \cdot 2$$

$$v_{\beta} = \sqrt{v_{\alpha}^2 - 2 g h_3}$$

$$\vec{T} = m \cdot \vec{a}$$

$$\vec{T} = m \cdot \frac{dv}{dt} \quad | \cdot dt$$

$$\vec{T} \cdot dt = m \cdot dv \quad | \int$$

$$\vec{T} \int dt = m \int dv$$

$$\vec{T} \cdot t = m \cdot v$$

$$\frac{1}{2} \vec{T}_{ix} \cdot t = D_{ave}$$

$$(-T - G_x) \tilde{r} = m v_D - m v_C \Rightarrow v_D = \frac{m v_C - (T + G_x) \tilde{r}}{m}$$

DE

9.1



9.1

$$\Delta E_K = \Delta W(x)$$

$$\frac{1}{2} \omega_{\text{E}}^2 - \frac{1}{2} \omega_{\text{D}}^2 - (-6x - 7 - \frac{7}{2}) \cdot 10 \text{€} / (14)$$

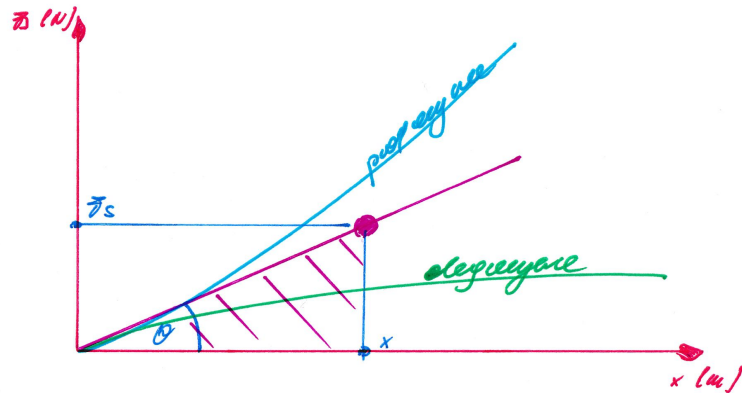
$$\rightarrow IE = \frac{m v_0^2}{2 (a_x + \delta + a)}$$

$$\underline{\Sigma F_{ix} = 0} \quad \Rightarrow \quad \underline{R - G = 0}$$

A diagram of a spring-mass system. A vertical spring is attached to a ceiling. A mass m is attached to the bottom of the spring. The equilibrium position is marked with a horizontal line. The displacement from equilibrium is labeled Δx . The coordinate system has x pointing down and l pointing up.

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$$\frac{1}{2} m v_0^2 = (Gx + J + \bar{F}) \cdot h$$



$$\frac{\bar{F}}{x} = \tan \theta$$

$$W = \frac{1}{2} \cdot \bar{F} \cdot x = \frac{1}{2} k \cdot x \cdot x = \frac{1}{2} k x^2$$

$$\Rightarrow \frac{1}{2} m v_0^2 = (mg \sin \beta + \eta mg \cos \beta + \frac{1}{2} k h^2) \cdot h$$

$$\frac{1}{2} m v_0^2 = mg \sin \beta \cdot h + \eta mg \cos \beta \cdot h + \frac{1}{2} k h^2$$

$$\frac{1}{2} k h^2 + (mg \sin \beta + \eta mg \cos \beta) h - \frac{1}{2} m v_0^2 = 0$$

